



HPC for dynamical analysis of elastic structures

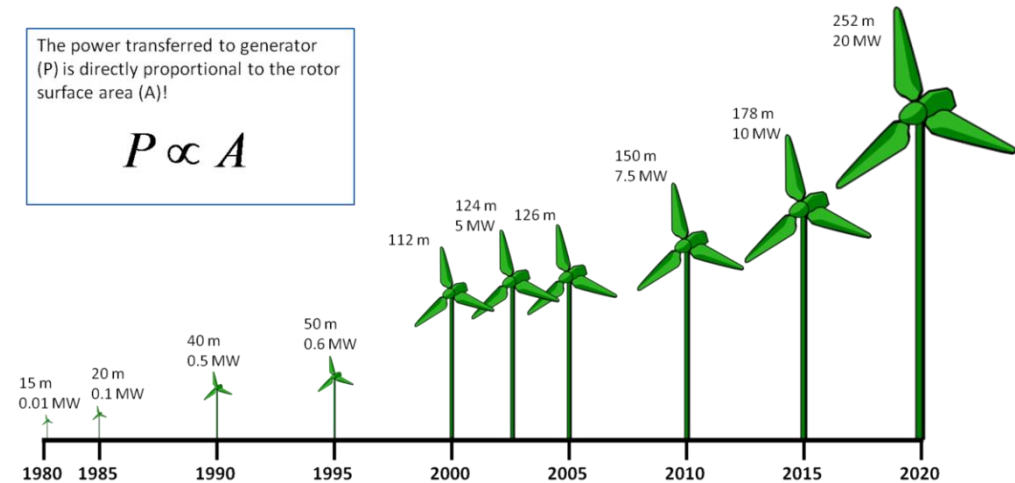
S. Stoykov



The tendency among the manufactures of modern engineering applications is to use more and more **computational mathematics** in the design, maintenance and health monitoring than laboratory experiments.

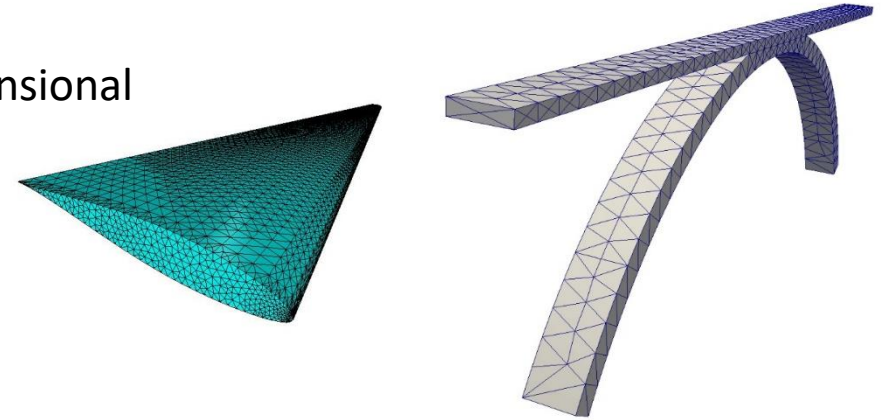
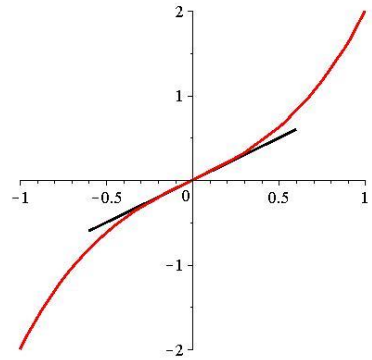
This can be achieved by

- More accurate physical models
- Better geometrical representation
- More efficient and accurate solvers
- Better understanding of the behavior of the structure due to variety of parameters



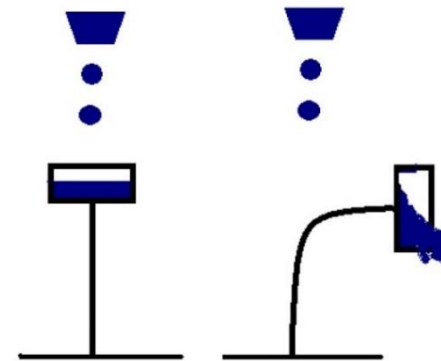
More accurate numerical results of complex structures require three-dimensional modelling

More accurate physical models require usage of nonlinear equations



Nonlinearity can

- Drastically change the dynamic behavior
- Multiple solutions for the same excitation frequency
- Extreme sensitivity to initial conditions
- Chaotic or quasi periodic motions



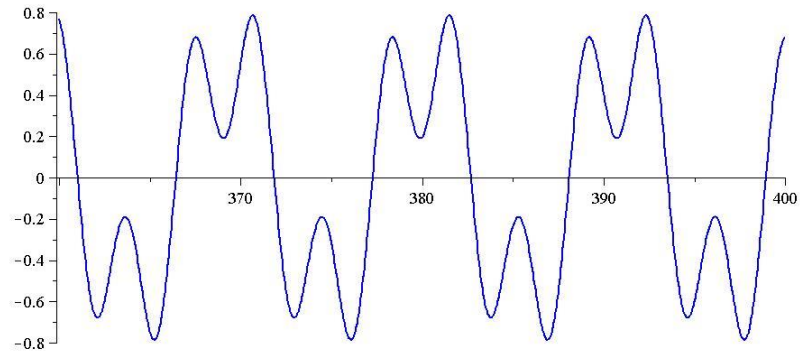
Nonlinearity occurs frequently in engineering applications.

Tacoma Narrows Bridge (1940) – collapsed because the possibility that *small periodic aerodynamic forces may become significant* **was not considered**.

The failure was due to insufficient torsional stiffness to resist large displacements.



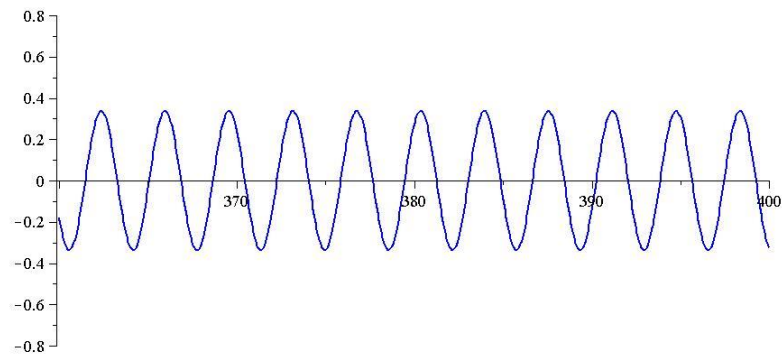
Small changes of **some of the parameters** lead to significantly different responses.



$$\ddot{x}(t) + 0.1 \dot{x}(t) + x(t)^3 = \sin(2\pi\mathbf{0.277}t)$$

$$x(0) = 0 \text{ m}$$

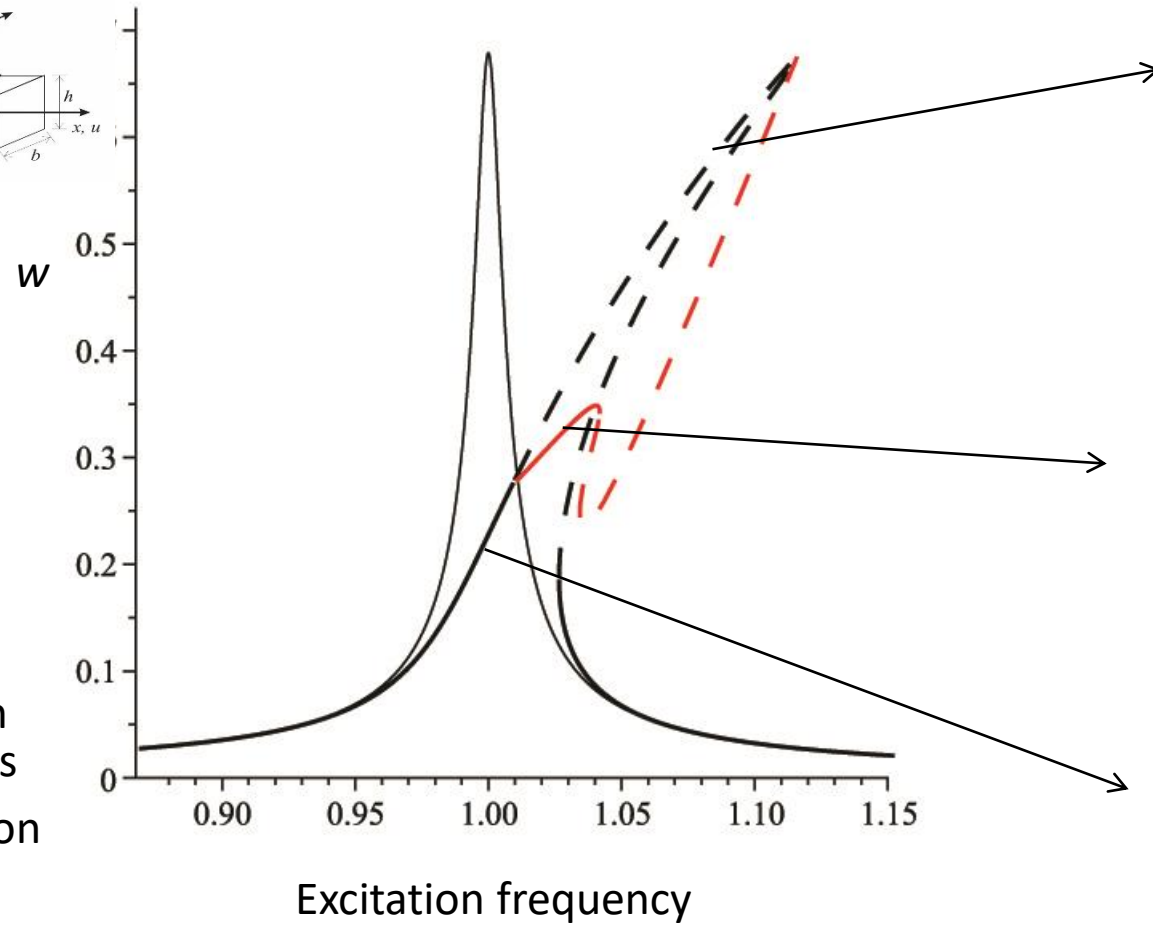
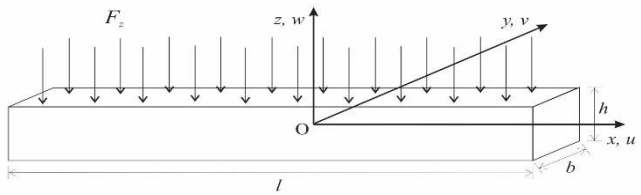
$$\dot{x}(0) = 0 \text{ m/s}$$



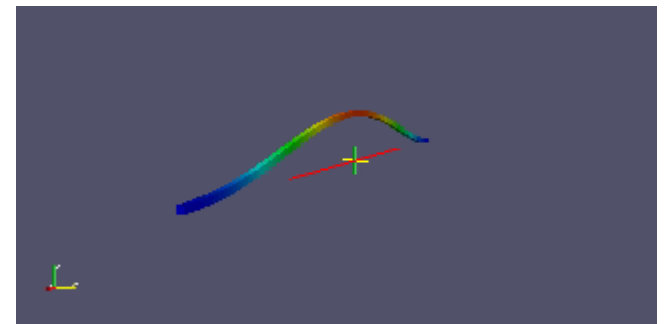
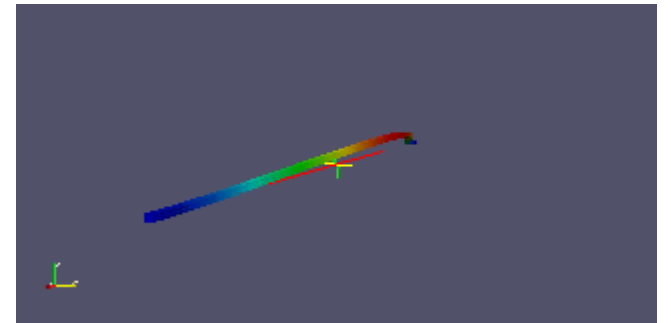
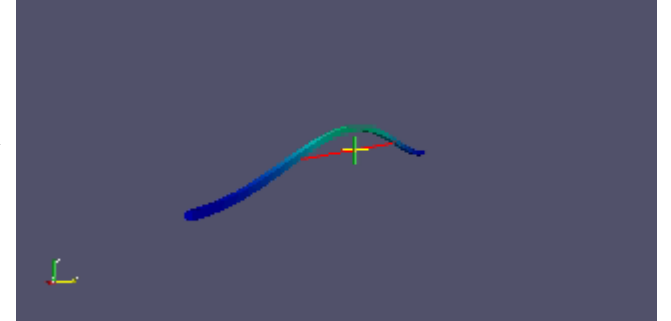
$$\ddot{x}(t) + 0.1 \dot{x}(t) + x(t)^3 = \sin(2\pi\mathbf{0.278}t)$$

$$x(0) = 0 \text{ m}$$

$$\dot{x}(0) = 0 \text{ m/s}$$

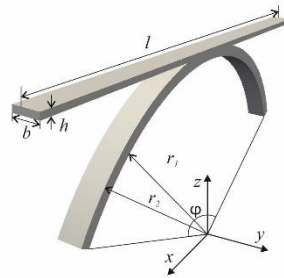


- Main branch
- Secondary branch
- - -** Unstable solutions
- w – amplitude of vibration

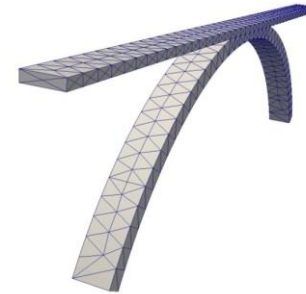
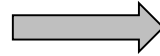




Engineering structure



Computer model



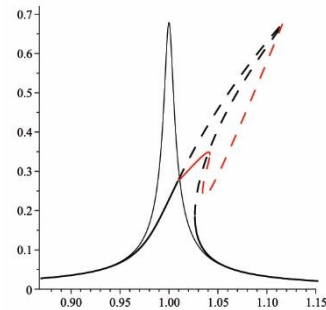
Finite element method

Equation of motion

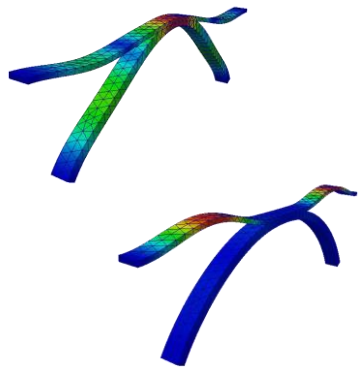
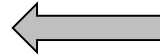
$$\rho_0 \frac{\partial^2 \mathbf{u}(\mathbf{x}, t)}{\partial t^2} - \nabla \cdot \mathbf{P} = \mathbf{f}$$



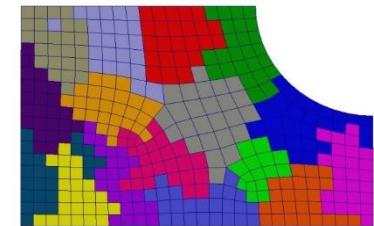
$$\mathbf{M} \ddot{\mathbf{q}}(t) + \mathbf{C} \dot{\mathbf{q}}(t) + \mathbf{K}(\mathbf{q}(t)) \mathbf{q}(t) = \mathbf{F}(t)$$

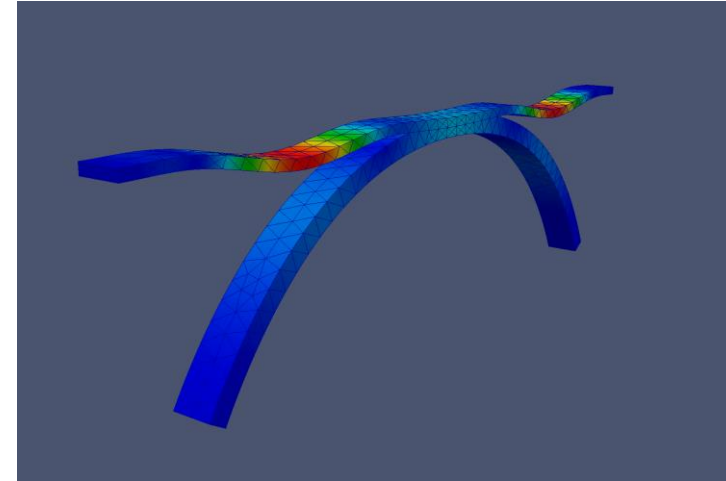
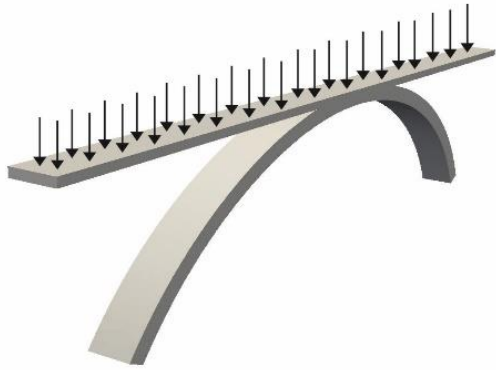


Frequency domain analysis

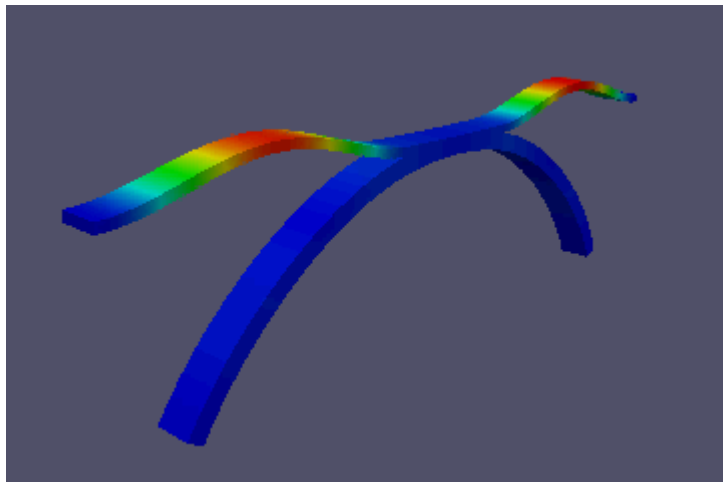


Shapes of vibration

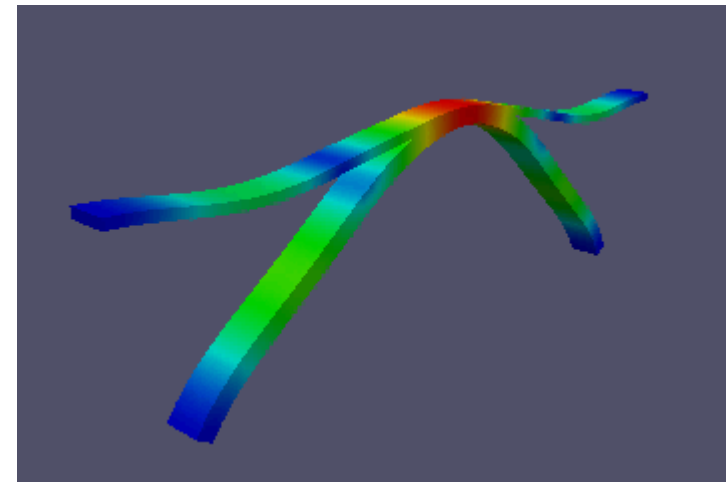




$\omega = 327$ rad/s



$\omega = 102$ rad/s



$\omega = 157$ rad/s



Thank you for your attention!

